



Miscellaneous open problems in the Regular Boundary Collocation approach

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Basic formulations on a simple example

Laplace equation with Dirichlet conditions

$$\nabla^2 \varphi(x, y) = 0 \quad (x, y) \in \Omega \quad (1)$$

$$\varphi(x, y) = f(x, y) \quad (x, y) \in \Gamma \quad (2)$$

$f(x, y)$ regular on Γ

Boundary
Value Problem

$$\varphi(x, y) \approx \hat{\varphi}(x, y) = \sum_k a_k \Phi_k(x, y) \equiv \mathbf{\Phi} \mathbf{a} \quad \mathbf{\Phi} \equiv \{\Phi_1, \Phi_2, \dots, \Phi_K\}, \quad \mathbf{a}^T \equiv \{a_1, a_2, \dots, a_K\} \quad (3)$$

Φ_k - Trefftz trial functions – fulfilling homogeneous equation inside Ω including Γ

Trefftz-type trial functions

$\Phi_k^H(x, y)$ - Herrera functions: complete sets of regular solutions of Eq.(1) (for open areas Ω singularity in central point $(0.0, 0.0)$)

$\Phi_k^K(x, y)$ - Kupradze functions: fundamental solutions with singularities outside Ω

$\Phi_k^O(x, y)$ - other Trefftz functions: e.g. fundamental solutions expanded into Fourier series or in fuzzy form (singularity extended to finite area)

Remark: functions $\Phi_k(x, y)$ singular on Γ (see BEM) in the Trefftz approach are excluded.

Open problems:

mixed-type Trefftz functions, location of singularities of fundamental solutions

Basic formulations on simple example

$$I = \int_{\Gamma} (\hat{\phi} - f)^2 d\Gamma = \int_{\Gamma} \left[\sum_k^K a_k \Phi_k - f \right]^2 d\Gamma = \min \quad (4)$$

$$\frac{\partial I}{\partial a_j} = 0 \quad \Rightarrow \quad \int_{\Gamma} \Phi_j (\hat{\phi} - f) d\Gamma = 0 \quad (5)$$

$j = 1, 2, 3 \dots K$

$$\sum_k^K a_k \int_{\Gamma} \Phi_j \Phi_k d\Gamma = \int_{\Gamma} \Phi_j f d\Gamma \quad (6)$$

$j = 1, 2, 3 \dots K$

simplest orthogonalization (other weighting functions possible)

Numerical integration

$$\int_{\Gamma} h(x, y) d\Gamma \Rightarrow \sum_L \int_{s_{1L}}^{s_{2L}} \tilde{h}(s) ds \Rightarrow \sum_L \sum_{\mu} \alpha_{\mu L} h(x_{\mu L}, y_{\mu L}) \stackrel{\text{notation}}{=} \sum_{\mu} [h(x, y)] \quad (7)$$

$(x_{\mu L}, y_{\mu L}) \in \Gamma$

$$\alpha_{\mu L} = \frac{s_{2L} - s_{1L}}{2} \tilde{\alpha}_{\mu}$$

L - segments of Γ

$\tilde{\alpha}_{\mu}$ - pure weights of integration

(x_{μ}, y_{μ}) - control points of integration on Γ

Numerical integral formulation

$$\int_{\Gamma} (\hat{\phi} - f)^2 d\Gamma = \min \quad \xRightarrow{\text{notation}} \quad \sum_{\mu}^{\Gamma} (\hat{\phi} - f)^2 = \min \quad (8)$$

$$\sum_k^K a_k \sum_{\mu}^M \alpha_{\mu} \Phi_{k\mu} \Phi_{j\mu} = \sum_{\mu}^M \alpha_{\mu} \Phi_{j\mu} f_{\mu} \quad j = 1, 2, 3, \dots \quad (9)$$

$$\Phi_{k\mu} \equiv \Phi_k(x_{\mu}, y_{\mu})$$

(x_{μ}, y_{μ}) - control points of integration on Γ

α_{μ} - weights of integration

More general: boundary collocation

$$\sum_{\mu}^{\Gamma} (\hat{\phi} - f)^2 = \min \quad (10)$$

$$\sum_k^K a_k \sum_{\nu}^M \beta_{\nu} \Phi_{k\nu} \Phi_{j\nu} = \sum_{\nu}^M \beta_{\nu} \Phi_{j\nu} f_{\nu} \quad j = 1, 2, \dots, K \quad \Phi_{k\nu} \equiv \Phi_k(x_{\nu}, y_{\nu}) \quad (11)$$

(x_{ν}, y_{ν}) - control points of collocation on Γ

β_{ν} - weights of collocation

Numerical boundary formulation

Open problem: choice of β_v and (x_v, y_v)

$\beta_v = \alpha_v$
 (x_v, y_v) - control points of integration on Γ } *Numerical integral –
specific case of boundary collocation*

Examples of control points of collocation along Γ :

- equidistant points,
- Gaussian points,
- Lobatto points.

Conclusion: proposed general name

Trefftz Method }
MFS } Regular Boundary Collocation Method

Specific case of boundary collocation: $K = M$

M - number of control points

Formal integral notation:

$$\int_{\Gamma} (\hat{\phi} - f) \delta(\mathbf{x} - \mathbf{x}_{\mu}) d\Gamma = 0 \quad (12) \quad \text{selecting property of } \delta(\mathbf{x} - \mathbf{x}_{\mu})$$

$\mu = 1, 2, 3 \dots M$

$$\beta_{\mu} \sum_k^K a_k \Phi_{k\mu} = \beta_{\mu} f_{\mu} \quad (x_{\mu}, y_{\mu}) \in \Gamma \quad (13)$$

$\mu = 1, 2, 3 \dots M$

Matrix notation for $K = M$:

$$\mathbf{B} \mathbf{a} = \mathbf{f}$$

\mathbf{B} - square matrix

Interpolation;

β_{μ} have no influence on results

Case: $M > K$

$$\mathbf{B}^T \mathbf{B} \mathbf{a} = \mathbf{B}^T \mathbf{f} \quad \text{least square} \quad 8$$

Integral case:

$$\int_{\Gamma} \Phi_k (\hat{\phi} - f) d\Gamma = 0 \quad (14)$$
$$k = 1, 2, 3 \dots K$$

M - number of integral control points

$M = K$ - interpolation; integral weights do not influence results

$M > K$ - overdetermined, least square

$M < K$ - approximation underdetermined

General form of collocation

Direct weighting

$$\sum_v^{\Gamma} \Phi_k(\hat{\phi} - f) = 0 \quad (15)$$
$$k = 1, 2, 3 \dots K$$

Opposite weighting

$$\sum_v^{\Gamma} \Phi_k^{\bullet}(\hat{\phi} - f) = 0 \quad (16)$$
$$k = 1, 2, 3 \dots K$$

$$\Phi_k^{\bullet} = \frac{\partial \Phi_k}{\partial n}$$

n - outward normal to Γ

Attention: $\Phi_k^H|_{k=1} = const \rightarrow$ necessary additional equation

Open problem: other types of weighting functions

Mixed Dirichlet-Neuman boundary conditions

$$\sum_{\mu}^{\Gamma_1} (\hat{\phi} - f)^2 + W^2 \sum_{\mu}^{\Gamma_2} \left(\frac{\partial \hat{\phi}}{\partial n} - g \right)^2 = \min \quad (17)$$

$$\sum_{\mu}^{\Gamma_1} \Phi_k (\hat{\phi} - f) + W^2 \sum_{\mu}^{\Gamma_2} \Phi_k \left(\frac{\partial \hat{\phi}}{\partial n} - g \right)^2 = 0 \quad (18)$$

$$k = 1, 2, 3 \dots K$$

Open problem: choice of W

Mixed Dirichlet-Neuman boundary conditions

$$\sum_{\mu}^{\Gamma_1} \Phi_k \left(\sum_i a_i \Phi_i - f \right) + W^2 \sum_{\mu}^{\Gamma_2} \Phi_k^{\bullet} \left(\sum_i a_i \Phi_i^{\bullet} - g \right) = 0 \quad (19)$$

$$k = 1, 2, 3, \dots, K$$

$$\Phi_k^{\bullet} = \frac{\partial \Phi_k}{\partial n}$$

$$\sum_i a_i \left[\underbrace{\sum_{\mu}^{\Gamma_1} \Phi_k \Phi_i}_{\text{terms 1}} + W^2 \underbrace{\sum_{\mu}^{\Gamma_2} \Phi_k^{\bullet} \Phi_i^{\bullet}}_{\text{terms 2}} \right] = \dots$$

$$k = 1, 2, 3, \dots, K$$

If terms 1 \gg terms 2, condition on Γ_2 will not be fulfilled

Opposite weighting:

$$\sum_{\mu}^{\Gamma_1} \Phi_k \cdot (\hat{\phi} - f) - \sum_{\mu}^{\Gamma_2} \Phi_k \left(\frac{\partial \hat{\phi}}{\partial n} - g \right) = 0 \quad (20)$$

$$k = 1, 2, \dots, K$$

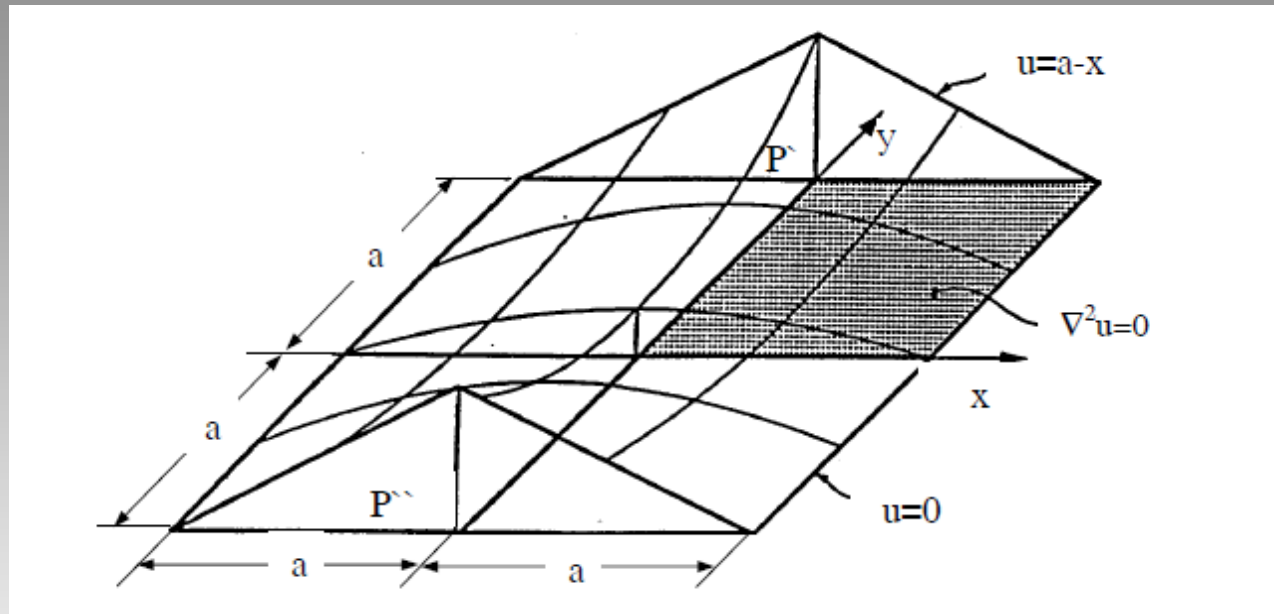
- weight not necessary
- symmetric stiffness matrix (approximately)
- possibility of negative definite matrices

$$\sum_{\mu}^{\Gamma_1} \Phi_k \cdot (\hat{\phi} - f) + \sum_{\mu}^{\Gamma_2} \Phi_k \left(\frac{\partial \hat{\phi}}{\partial n} - g \right) = 0 \quad (21)$$

$$k = 1, 2, \dots, K$$

P. Ladeveze, Ch.Hochard
- nonsymmetric matrices

Numerical example

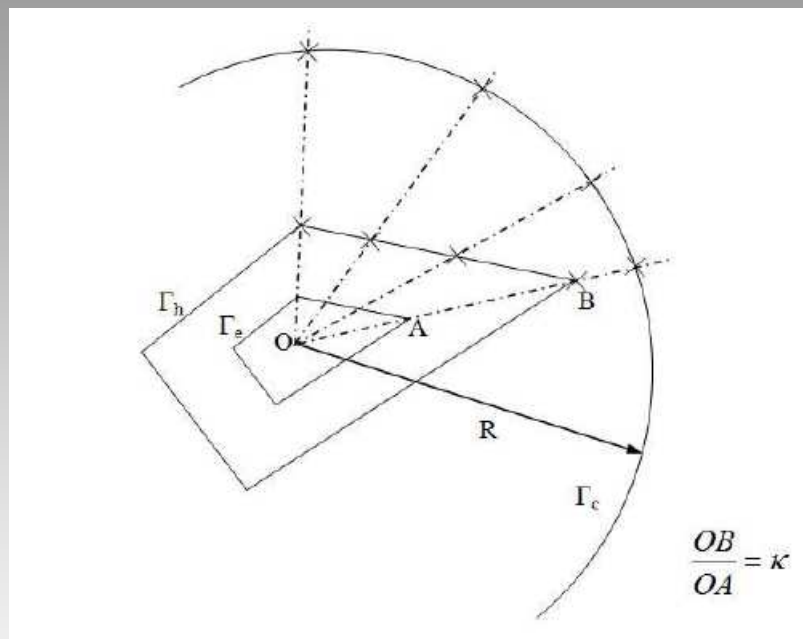


Singular membrane problem.

$$u(x, y) = \sum_{n=0}^N \frac{8a \cos[(2n+1)\pi x/(2a)] \cosh[(2n+1)\pi y/(2a)]}{(2n+1)^2 \pi^2 \cosh[(2n+1)\pi/2]} \quad (22)$$

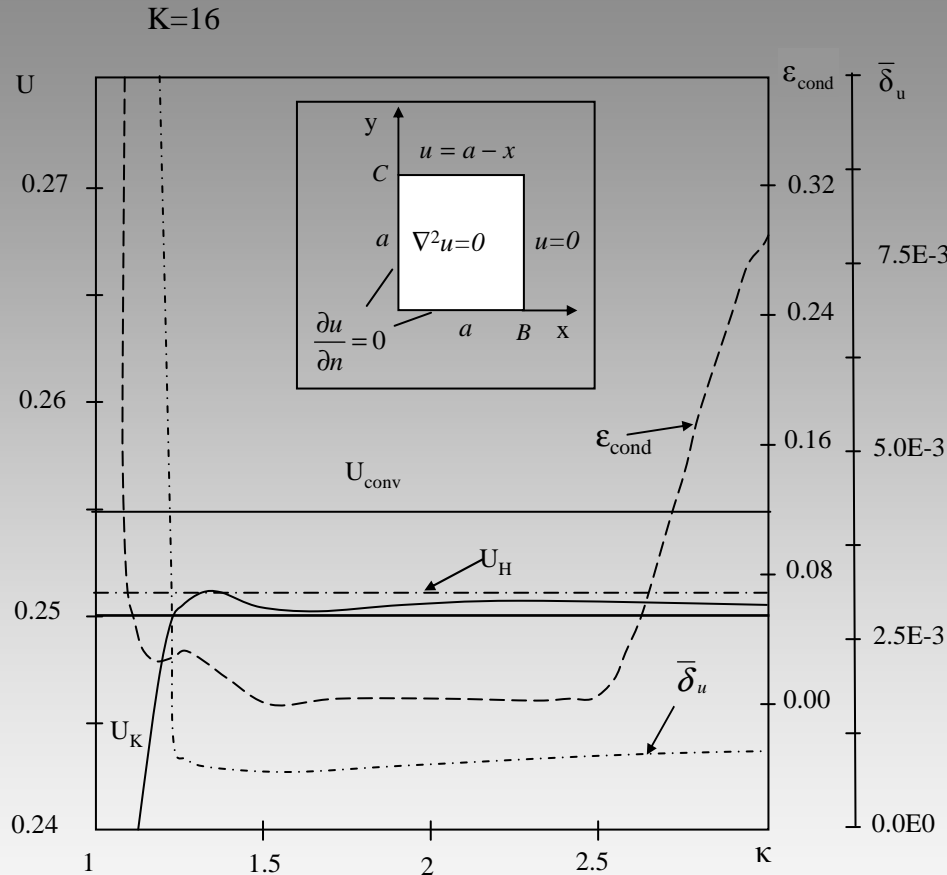
$N \rightarrow \infty$ (in the example $N = 200$)

Numerical example: singular membrane problem.



Position of singularities of fundamental solutions (Kupradze functions)

Numerical example: singular membrane problem.



$$\delta_u = \frac{u^{K,H} - u^{EX}}{u_c^{EX}}$$

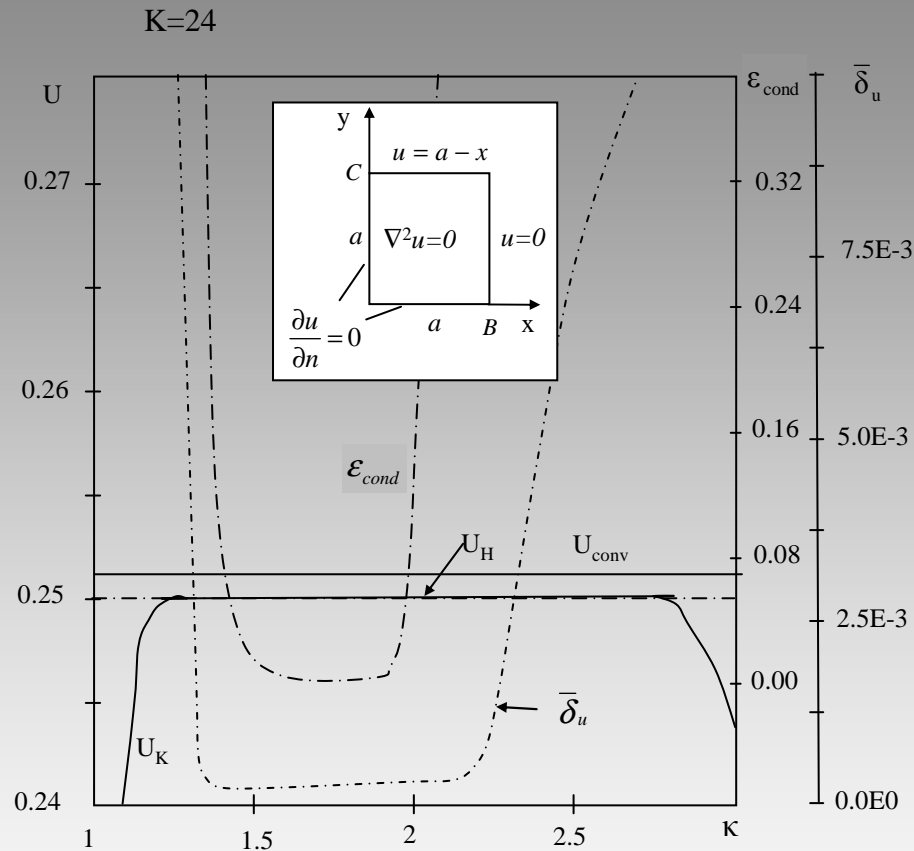
$$\bar{\delta}_u = \frac{\sum_{i,j=1}^N |u_{i,j}^{K,H} - u_{i,j}^{EX}|}{N^2 u_c^{EX}} = \frac{1}{N^2} \sum_{i,j=1}^N |\delta_u|_{ij}$$

$$U^{EX} = \frac{1}{2} \int_{\Omega} \left[\left(\frac{\partial u^{EX}}{\partial x} \right)^2 + \left(\frac{\partial u^{EX}}{\partial y} \right)^2 \right] d\Omega = 0.25a^2$$

$$\epsilon_{\text{cond}} = \max_i |(\mathbf{H}^{-1}\mathbf{H})_{ii} - 1|$$

Range of acceptable distance from boundary Γ_c of singularities of the Kupradze functions: homothetic contour Γ_h . Generalized energy calculated with help of a single conventional p-element (U_{conv}), HT-H approach (U_H), and HT-K formulation (U_K). $a = 1$, $U^{EX} = 0.25$

Numerical example: singular membrane problem.



$$\delta_u = \frac{u^{K,H} - u^{EX}}{u_c^{EX}}$$

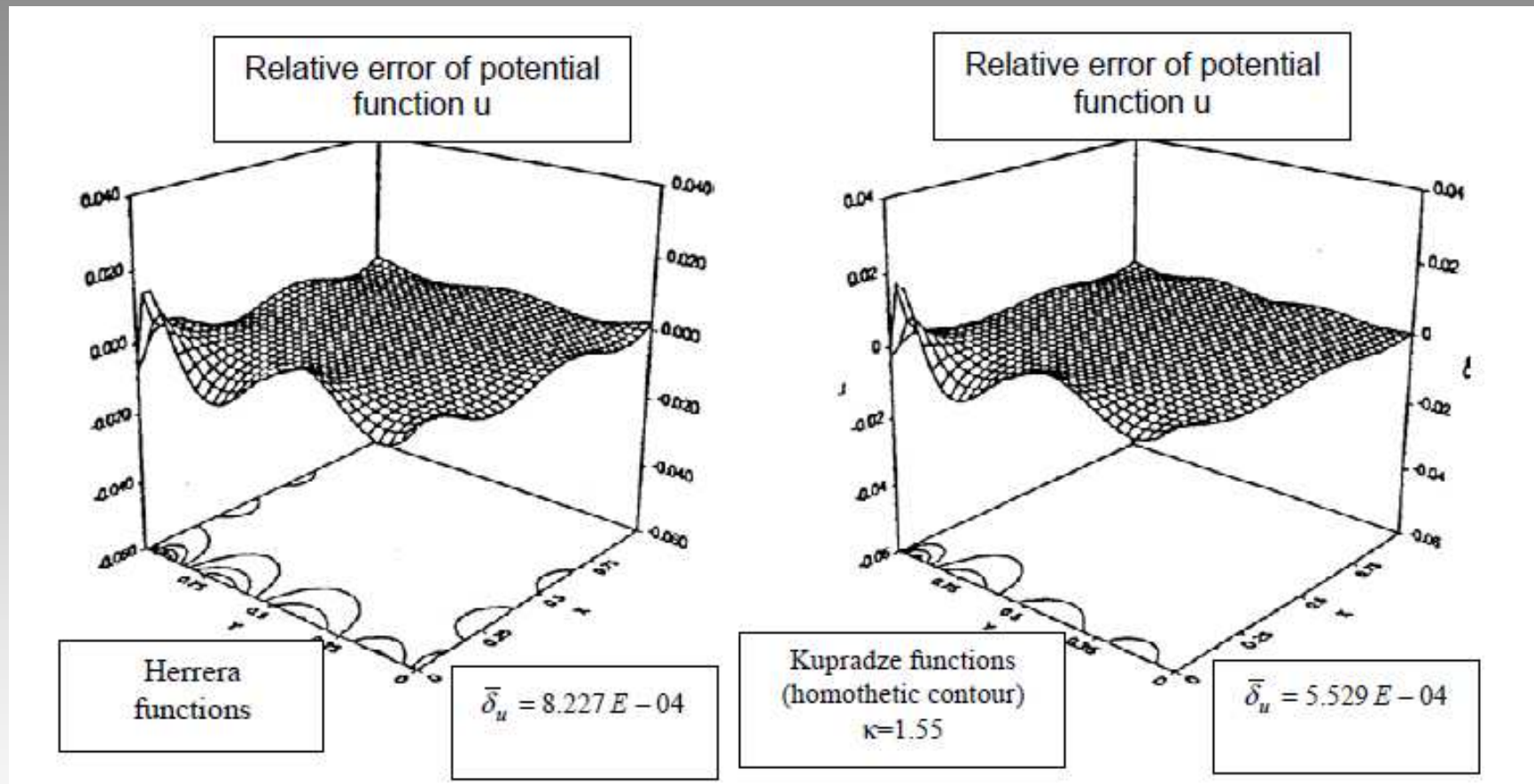
$$\bar{\delta}_u = \frac{\sum_{i,j=1}^N |u_{i,j}^{K,H} - u_{i,j}^{EX}|}{N^2 u_c^{EX}} = \frac{1}{N^2} \sum_{i,j=1}^N |\delta_u|_{ij}$$

$$U^{EX} = \frac{1}{2} \int_{\Omega} \left[\left(\frac{\partial u^{EX}}{\partial x} \right)^2 + \left(\frac{\partial u^{EX}}{\partial y} \right)^2 \right] d\Omega = 0.25a^2$$

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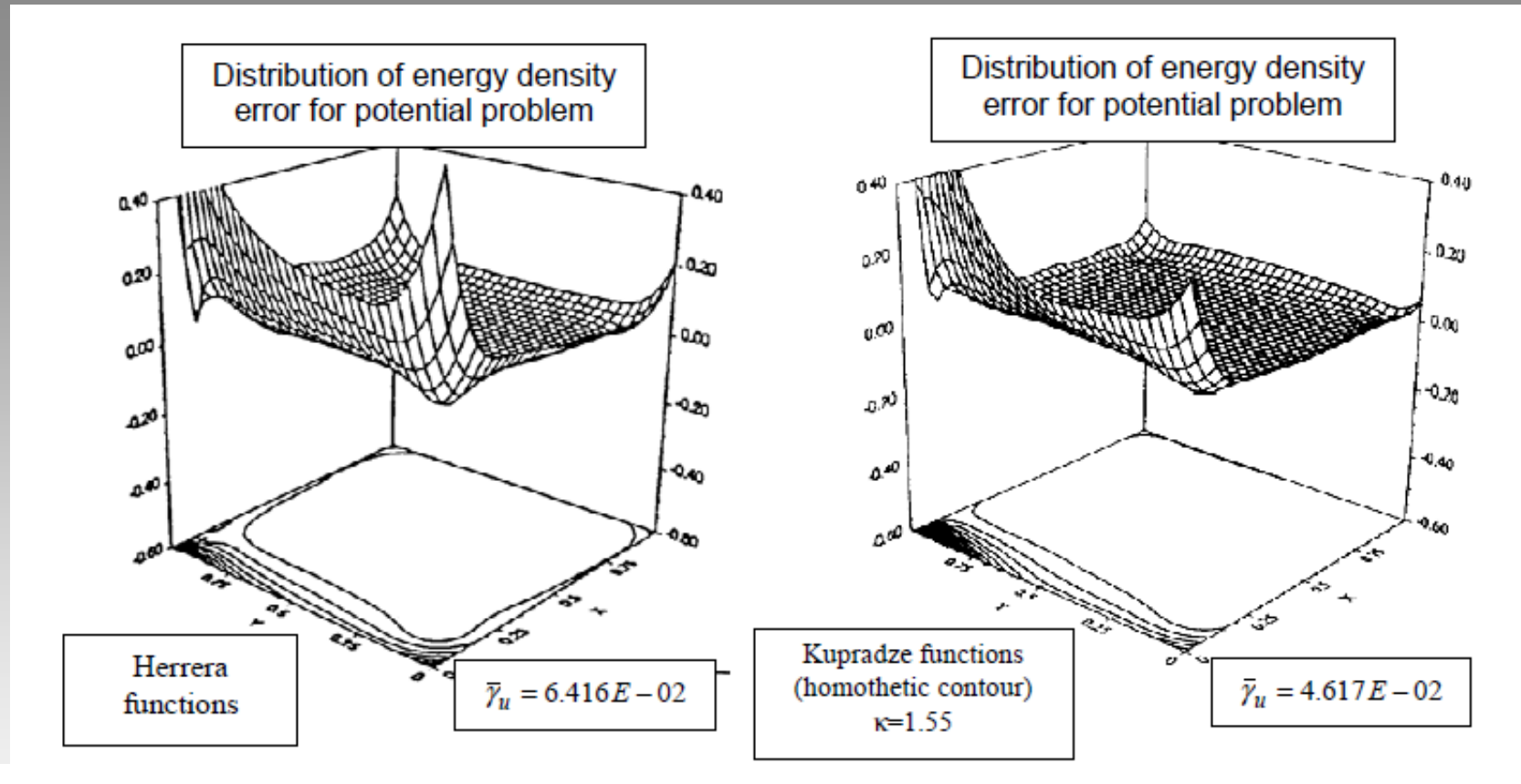
Range of acceptable distance from boundary Γ_s of singularities of the Kupradze functions; homothetic contour Γ_h . Generalized energy calculated with help of a single conventional p-element (U_{conv}), HT-H approach (U_H), and HT-K formulation (U_K). $a=1$, $U^{EX}=0.25$, $u_c^{EX}=1.0$

Numerical example: singular membrane problem.



$$\bar{\delta}_u = \frac{\sum_{i,j=1}^N |u_{i,j}^{K,H} - u_{i,j}^{EX}|}{N^2 u_c^{EX}} = \frac{1}{N^2} \sum_{i,j=1}^N |\delta_u|_{ij}$$

Numerical example: singular membrane problem.

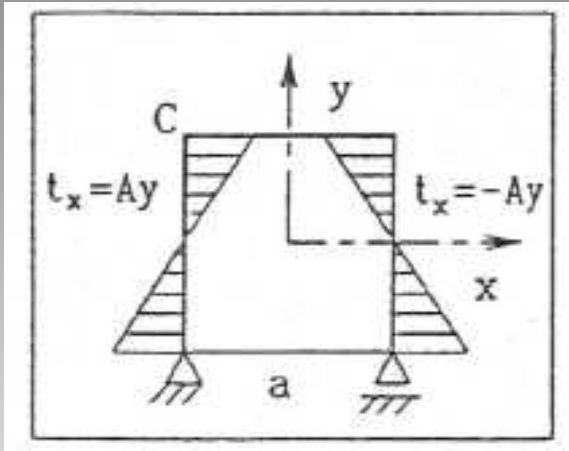


$$\gamma_U = \left\{ \frac{\left[\frac{\partial}{\partial x} (u^{K,H} - u^{EX}) \right]^2 + \left[\frac{\partial}{\partial y} (u^{K,H} - u^{EX}) \right]^2}{2\rho^{EX}} \right\}^{1/2}$$

$$\rho^{EX} = \frac{U^{EX}}{a^2} = 0.25$$

$$\bar{\gamma}_U = \frac{1}{N^2} \sum_{i,j=1}^N \gamma_{Uij}$$

Numerical example: 2D elasticity



$$u^{EX}(x, y) = \frac{A}{E}(1 - xy), \quad v^{EX}(x, y) = \frac{A}{2E}(x^2 + \nu y^2 - \nu - 1) .$$

$$\delta_u = \left\{ \frac{(u^{K,H} - u^{EX})^2 + (v^{K,H} - v^{EX})^2}{(u_C^{EX})^2 + (v_C^{EX})^2} \right\}^{1/2}$$

$$\bar{\delta}_u = \frac{1}{N^2} \sum_{i,j=1}^N \delta_{u ij}$$

Energy density error \rightarrow

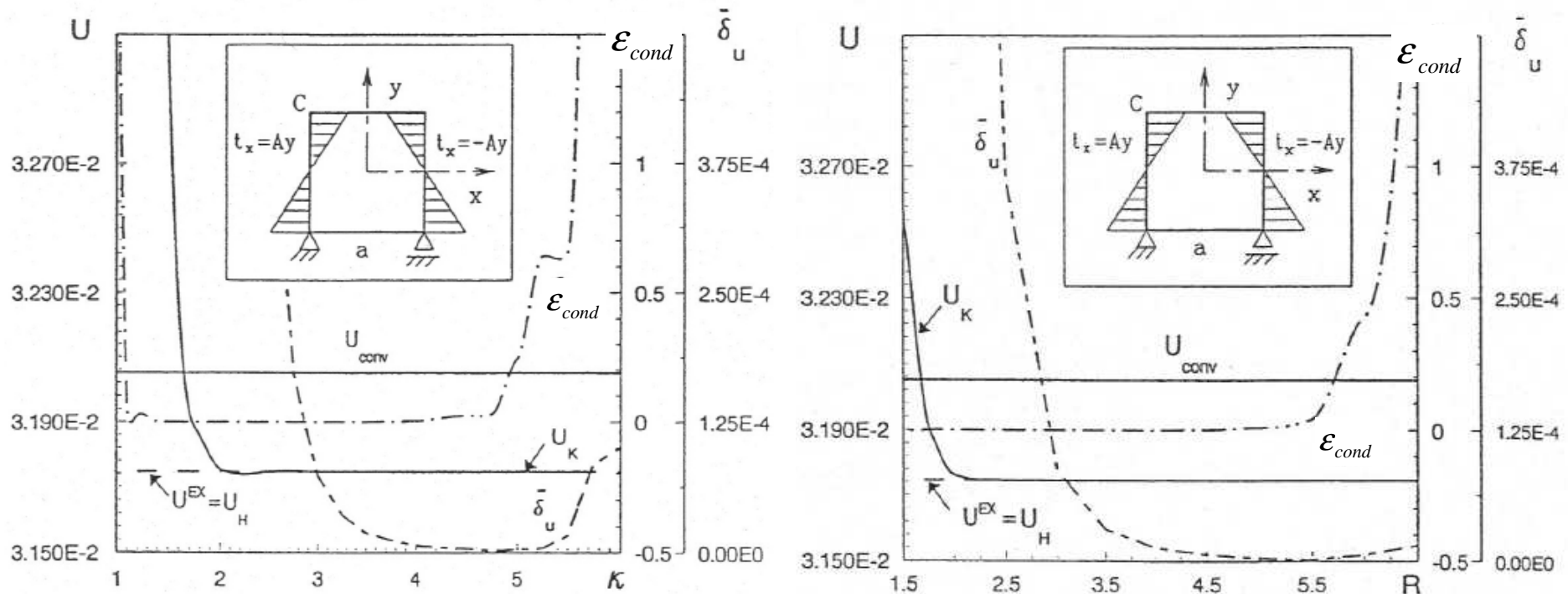
$$\gamma_U = \left\{ \frac{1}{E\rho^{EX}} \left[\frac{1}{2}(\sigma_x^{K,H} - \sigma_x^{EX})^2 + \frac{1}{2}(\sigma_y^{K,H} - \sigma_y^{EX})^2 - \nu(\sigma_x^{K,H} - \sigma_x^{EX})(\sigma_y^{K,H} - \sigma_y^{EX}) + (1+\nu)(\tau_{xy}^{K,H} - \tau_{xy}^{EX})^2 \right] \right\}^{1/2}, \quad \rho^{EX} = \frac{U^{EX}}{a^2}$$

$$\bar{\gamma}_U = \frac{1}{N^2} \sum_{i,j=1}^N \gamma_{U ij}$$

E - Young modulus

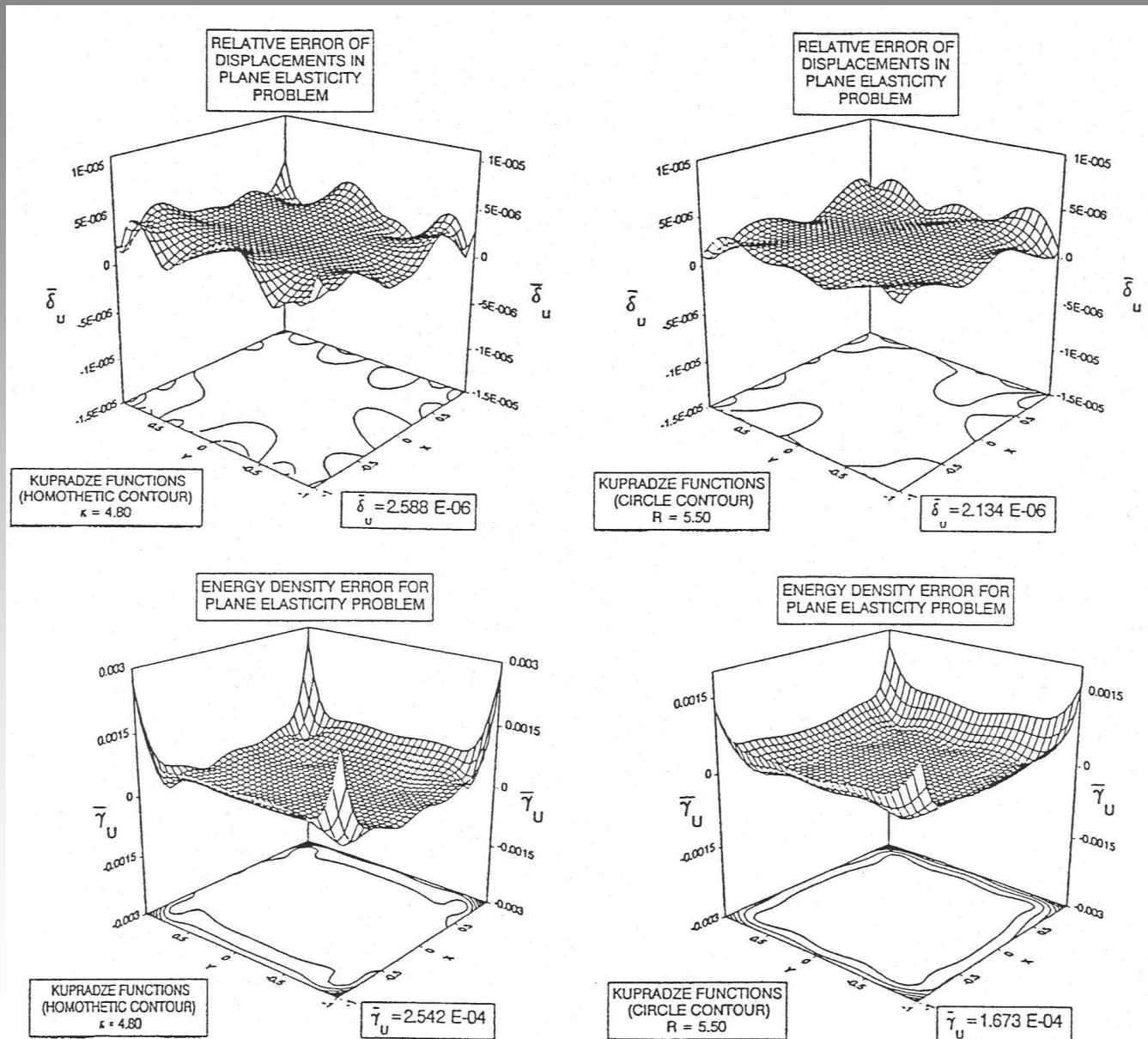
ν - Poisson ratio

Numerical example: 2D elasticity



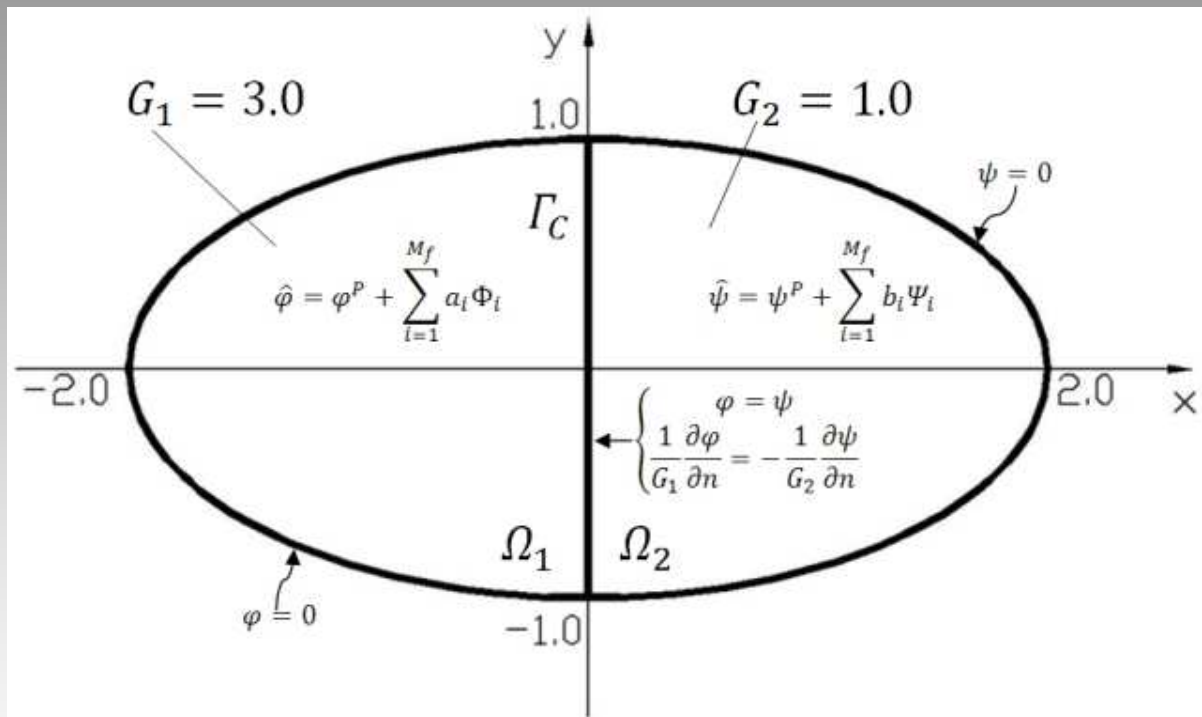
Range of acceptable distance from the element boundary of singularities of Kupradze functions in HT-K solutions. Kupradze functions with singularities situated (a) on a homothetic contour and (b) on a circle: $K = 12$, $a = 2$, $A = 100$. Generalized energy calculated with help of a single conventional p-element (U_{conv}), HT-H approach (U_H), and HT-K formulation (U_K), $U^{EX} = 0.031746$.

Numerical example: 2D elasticity



Coupling of two subregions

Torsion of a bar made of different materials



Cross-section of an elliptic bar made of two different materials – example

Coupling of two subregions

Torsion of a bar made of different materials

$$\nabla^T \frac{1}{G_1} \nabla \phi = -2\beta \quad (x, y) \in \Omega_1$$

$$\nabla^T \frac{1}{G_2} \nabla \psi = -2\beta \quad (x, y) \in \Omega_2$$

$$\phi = 0 \quad (x, y) \in \Gamma_1$$

$$\psi = 0 \quad (x, y) \in \Gamma_2$$

$$\left. \begin{array}{l} \phi = \psi \\ \frac{1}{G_1} \frac{\partial \phi}{\partial n} = -\frac{1}{G_2} \frac{\partial \psi}{\partial n} \end{array} \right\} \quad (x, y) \in \Gamma_c$$

β - angle of twist per unit bar length

ϕ, ψ - Prandtl functions

G_1, G_2 - Kirhchoff moduli

Coupling of two subregions

Torsion of a bar made of different materials

$$\int_{\Gamma_1} W_j^{(1)} \hat{\phi} d\Gamma_1 + \int_{\Gamma_c} \left[W_j^{(2)} (\hat{\phi} - \hat{\psi}) + W_j^{(3)} \left(\frac{1}{G_1} \hat{\phi} \cdot + \frac{1}{G_2} \hat{\psi} \cdot \right) \right] d\Gamma_c = 0$$

$$j = 1, 2, \dots, K$$

$$\int_{\Gamma_c} \left[W_j^{(4)} \left(\frac{1}{G_2} \hat{\psi} \cdot + \frac{1}{G_1} \hat{\phi} \cdot \right) + W_j^{(5)} (\hat{\psi} - \hat{\phi}) \right] d\Gamma_c + \int_{\Gamma_2} W_j^{(6)} \hat{\psi} d\Gamma_2 = 0$$

$$\hat{\phi} = \sum_k a_k^1 \Phi_k + \phi^p$$

$$\hat{\psi} = \sum_k a_k^2 \Psi_k + \psi^p$$

$$\hat{\phi} \cdot \equiv \frac{\partial \hat{\phi}}{\partial n} = \sum_k a_k^1 \Phi_k \cdot + \phi^{\cdot p}$$

$$\hat{\psi} \cdot \equiv \frac{\partial \hat{\psi}}{\partial n} = \sum_k a_k^2 \Psi_k \cdot + \psi^{\cdot p}$$

ϕ^p, ψ^p – particular solutions of nonhomogeneous equations

Coupling of two subregions

Torsion of a bar made of different materials

$$W_j^{(1)} = W_j^{(2)} = \Phi_j \qquad W_j^{(3)} = \frac{l}{G_1} \Phi_j^\bullet$$

$$W_j^{(5)} = W_j^{(6)} = \Psi_j \qquad W_j^{(4)} = \frac{l}{G_2} \Psi_j^\bullet$$

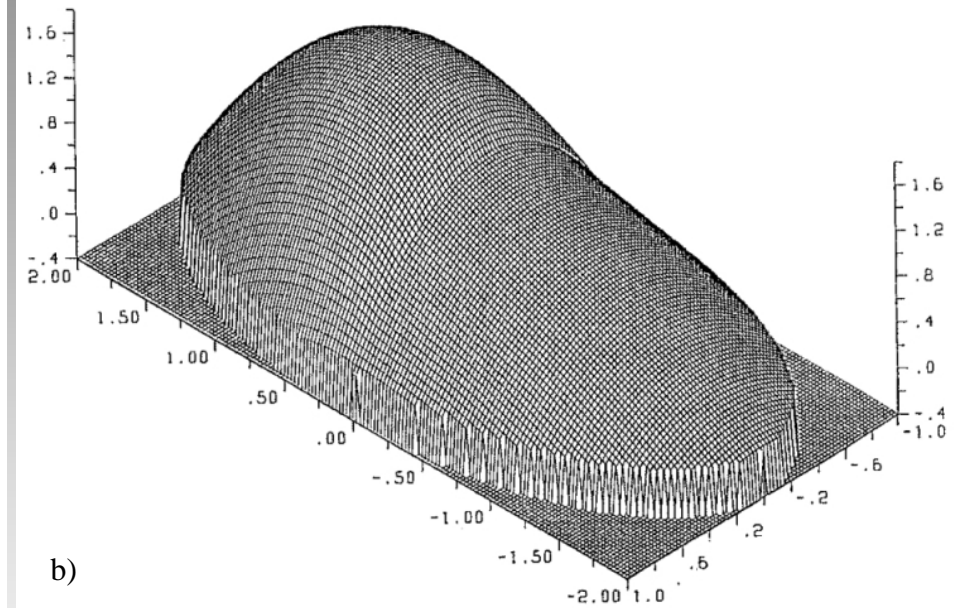
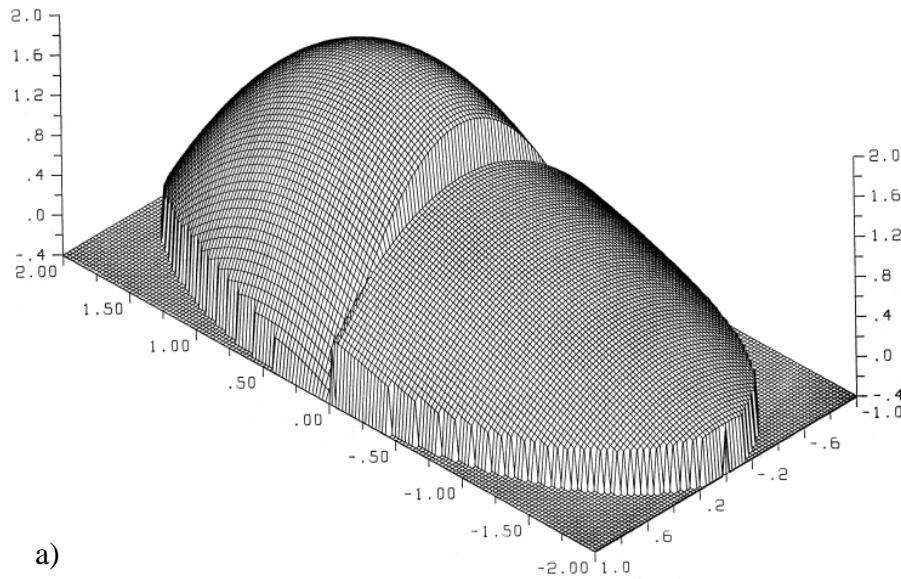
$$\mathbf{K} \mathbf{a} = \mathbf{f}$$

$$K_{jk} = \left\| \begin{array}{c|c} \int_{\Gamma_{l+c}} \Phi_j \Phi_k d\Gamma_{l+c} + \frac{l}{G_1^2} \int_{\Gamma_c} \Phi_j^\bullet \Phi_k^\bullet d\Gamma_c & - \int_{\Gamma_c} \left(\Phi_j \Psi_k - \frac{l}{G_1 G_2} \Phi_j^\bullet \Psi_k^\bullet \right) d\Gamma_c \\ \hline - \int_{\Gamma_c} \left(\Psi_j \Phi_k - \frac{l}{G_1 G_2} \Psi_j^\bullet \Phi_k^\bullet \right) d\Gamma_c & \int_{\Gamma_{2+c}} \Psi_j \Psi_k d\Gamma_{2+c} + \frac{l}{G_2^2} \int_{\Gamma_c} \Psi_j^\bullet \Psi_k^\bullet d\Gamma_c \end{array} \right\|$$

$$j, k = 1, 2, \dots, K$$

Coupling of two subregions

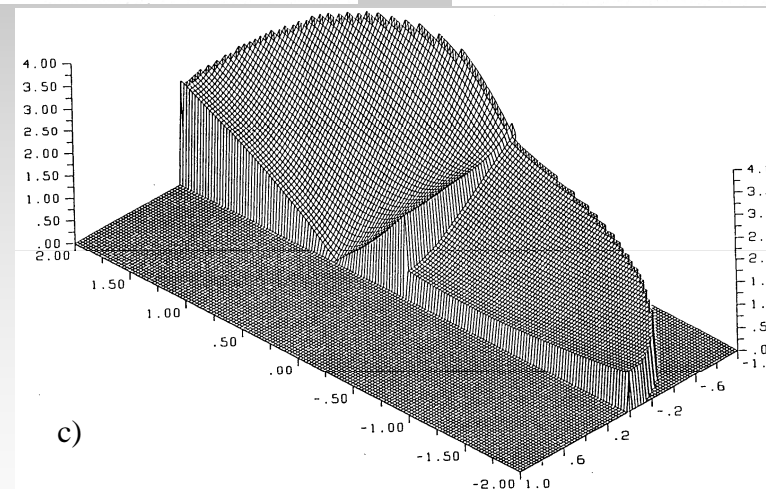
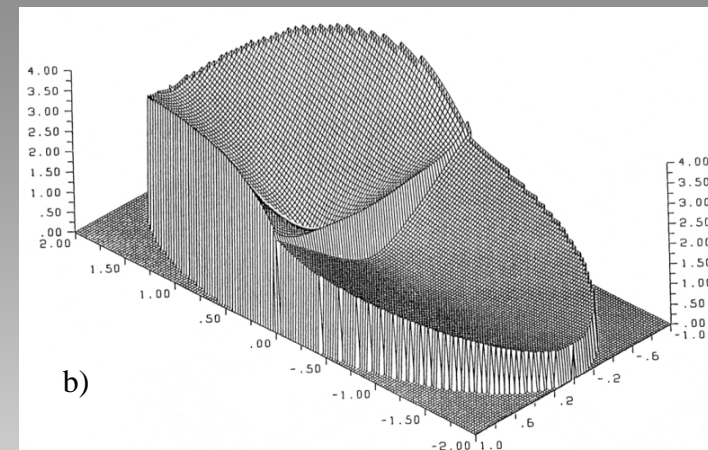
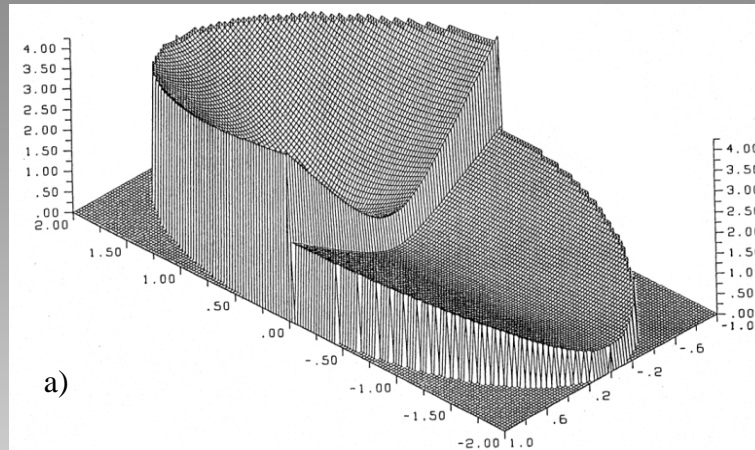
Torsion of a bar made of different materials



Direct coupling of the two half-elliptic regions – stress function, $G_1=3.0$, $G_2=1.0$,
Herrera functions, a) $K=3$; b) $K=9$;

Coupling of two subregions

Torsion of a bar made of different materials



$$\tau = \left[\left(\frac{\partial \varphi}{\partial x} \right)^2 + \left(\frac{\partial \varphi}{\partial y} \right)^2 \right]^{1/2}$$

Direct coupling of the two half-elliptic regions – resulting stresses τ , $G1 = 3.0$, $G2 = 1.0$:

Herrera functions,

a) $K = 3$;

b) $K = 9$;

c) $K = 9$ – cross-section

Coupling of many subregions (elements) – J. Jirousek 1994

Example: Laplace equation

$$J(\mathbf{a}) = \int_{\Gamma_\varphi} (\hat{\varphi} - \bar{\varphi})^2 d\Gamma + W_{qn}^2 \int_{\Gamma_{qn}} [(\hat{\varphi})'_n - (\bar{\varphi})'_n]^2 d\Gamma +$$

$$+ \sum_I \left\{ \int_{\Gamma_I} (\hat{\varphi}^+ - \hat{\varphi}^-)^2 d\Gamma + W_I^2 \int_{\Gamma_I} [(\hat{\varphi}^+)'_n + (\hat{\varphi}^-)'_n]^2 d\Gamma \right\} = \min$$

$\hat{\varphi}(x, y)$ - Trefftz solution in each subregion

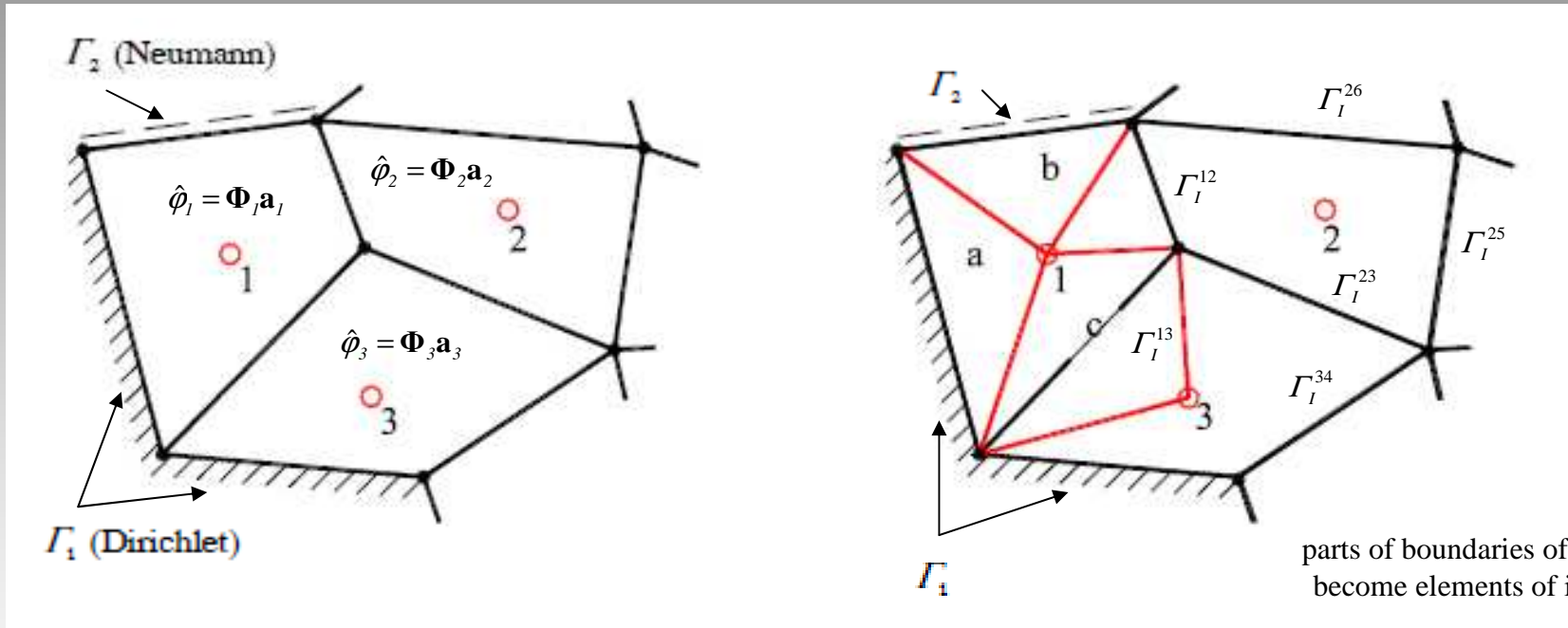
$\bar{\varphi}$ - given on $\Gamma_\varphi \equiv \Gamma_1$

$(\bar{\varphi})'_n$ - given on $\Gamma_{qn} \equiv \Gamma_2$

$$\varphi(x, y) \approx \hat{\varphi}(x, y) = \sum_k a_k \Phi_k(x, y) \equiv \mathbf{\Phi} \mathbf{a} \quad \mathbf{\Phi} \equiv \{\Phi_1, \Phi_2, \dots, \Phi_K\}, \quad \mathbf{a}^T \equiv \{a_1, a_2, \dots, a_K\}$$

Φ_k - Trefftz trial functions – fulfilling homogeneous equation inside Ω including Γ

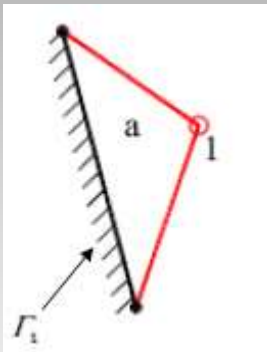
Coupling of many subregions



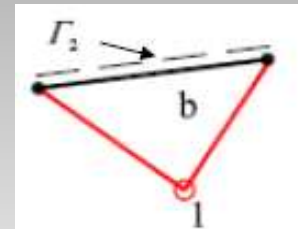
Coupling of many subregions

$$\delta J(\mathbf{a}) = \delta \mathbf{a}^T \frac{\partial J}{\partial \mathbf{a}} = \delta \mathbf{a}^T (\mathbf{K}\mathbf{a} - \mathbf{R}) = 0$$

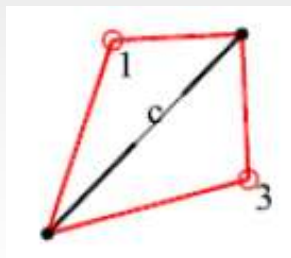
$$\delta J(\mathbf{a}) = \sum_{e=1}^{N_e} \delta J_e(\mathbf{a}_e) \quad \delta J_e(\mathbf{a}_e) = \delta \mathbf{a}_e^T \frac{\partial J_e(\mathbf{a}_e)}{\partial \mathbf{a}_e} = \delta \mathbf{a}_e^T (\mathbf{r}_e^p + \mathbf{k}_e \mathbf{a}_e) = \delta \mathbf{a}_e^T \mathbf{r}_e \longrightarrow \mathbf{r}_e = \mathbf{r}_e^p + \mathbf{k}_e \mathbf{a}_e$$



$$\mathbf{k}_a = \sum_{\mu}^{\Gamma_1} \Phi_1^T \Phi_1$$



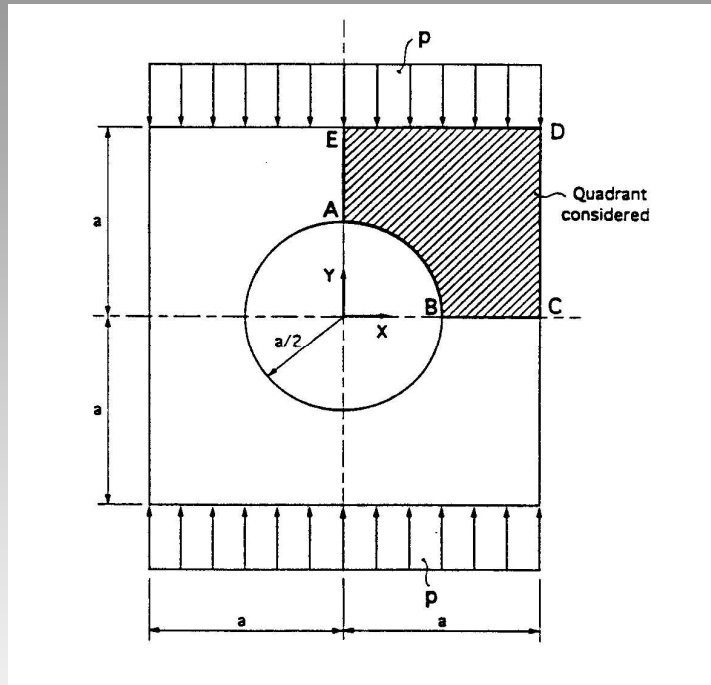
$$\mathbf{k}_b = W^2 \sum_{\mu}^{\Gamma_2} \Phi_1^{\bullet T} \Phi_1^{\bullet}$$



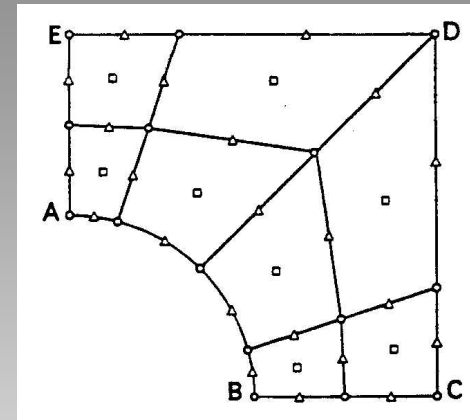
$$\mathbf{k}_c = \begin{bmatrix} \sum_{\mu}^{\Gamma_{13}} [\Phi_1^T \Phi_1 + W^2 \Phi_1^{\bullet T} \Phi_1^{\bullet}] & \sum_{\mu}^{\Gamma_{13}} [-\Phi_1^T \Phi_3 + W^2 \Phi_1^{\bullet T} \Phi_3^{\bullet}] \\ \sum_{\mu}^{\Gamma_{13}} [-\Phi_3^T \Phi_1 + W^2 \Phi_3^{\bullet T} \Phi_1^{\bullet}] & \sum_{\mu}^{\Gamma_{13}} [\Phi_3^T \Phi_3 + W^2 \Phi_3^{\bullet T} \Phi_3^{\bullet}] \end{bmatrix}$$

Numerical example (J. Jirousek, A. Wróblewski, 1994):

Compression of a perforated panel

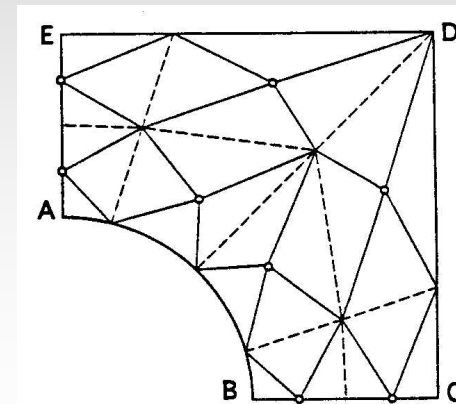


Conventional p-element mesh



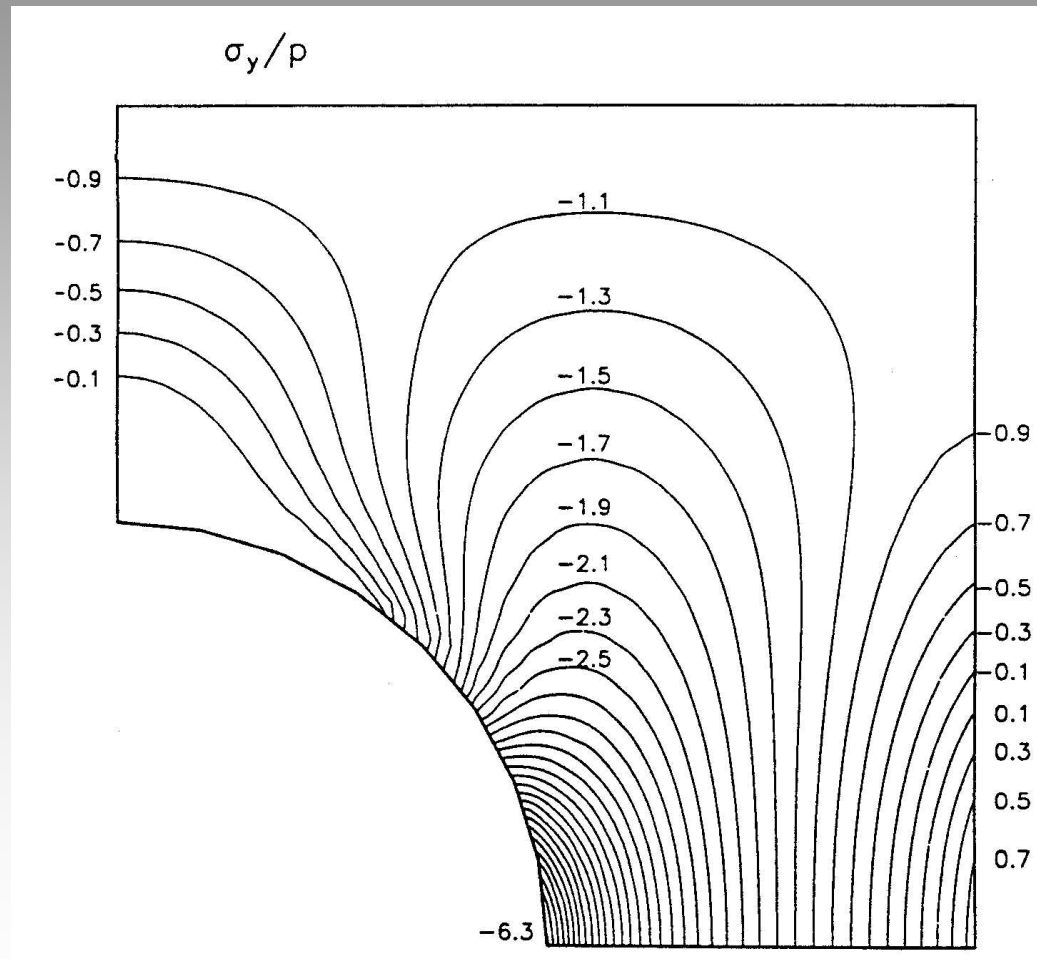
232 DOF
($p=5$)

T-element mesh

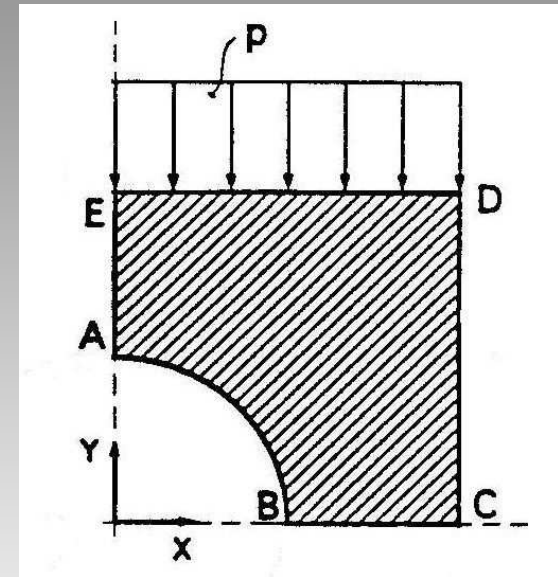
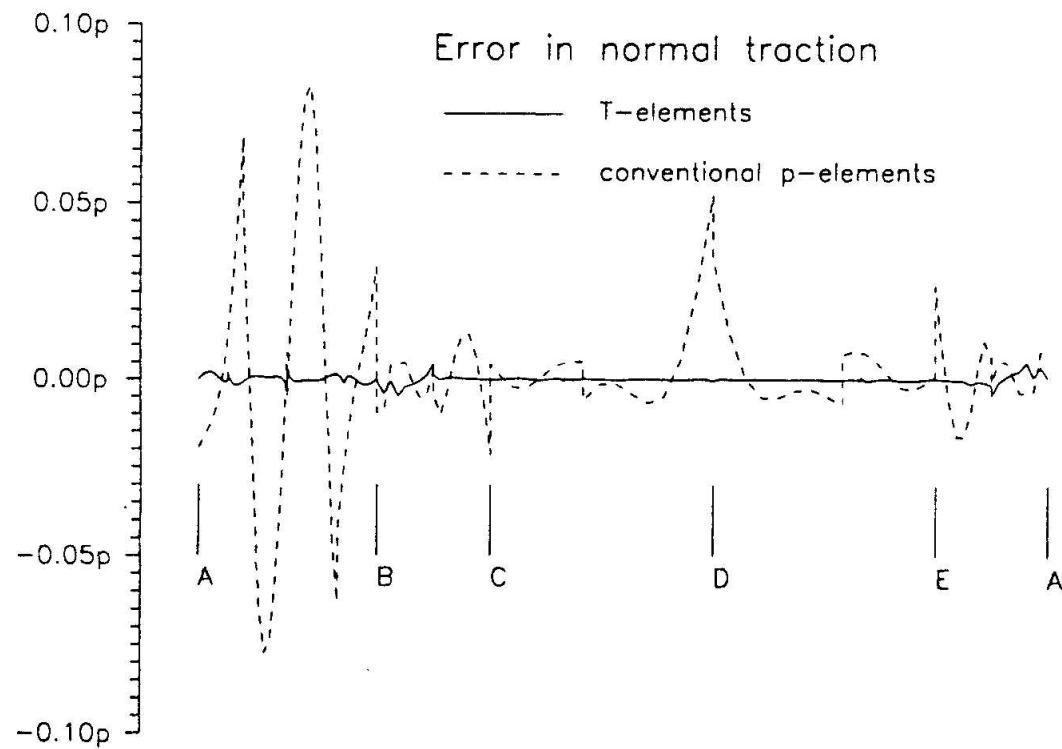


228 DOF
($p=9$)

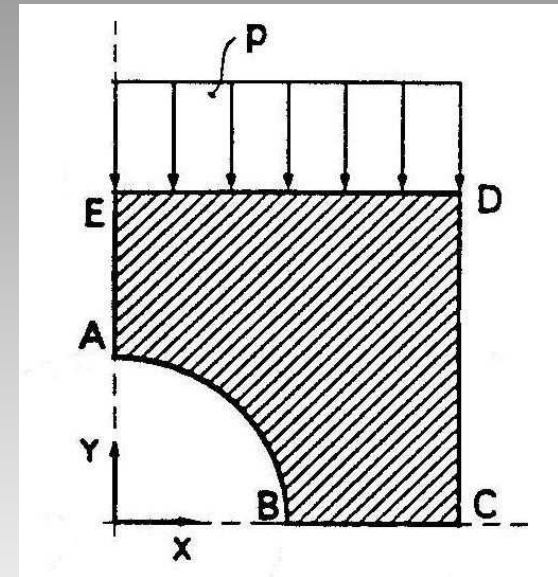
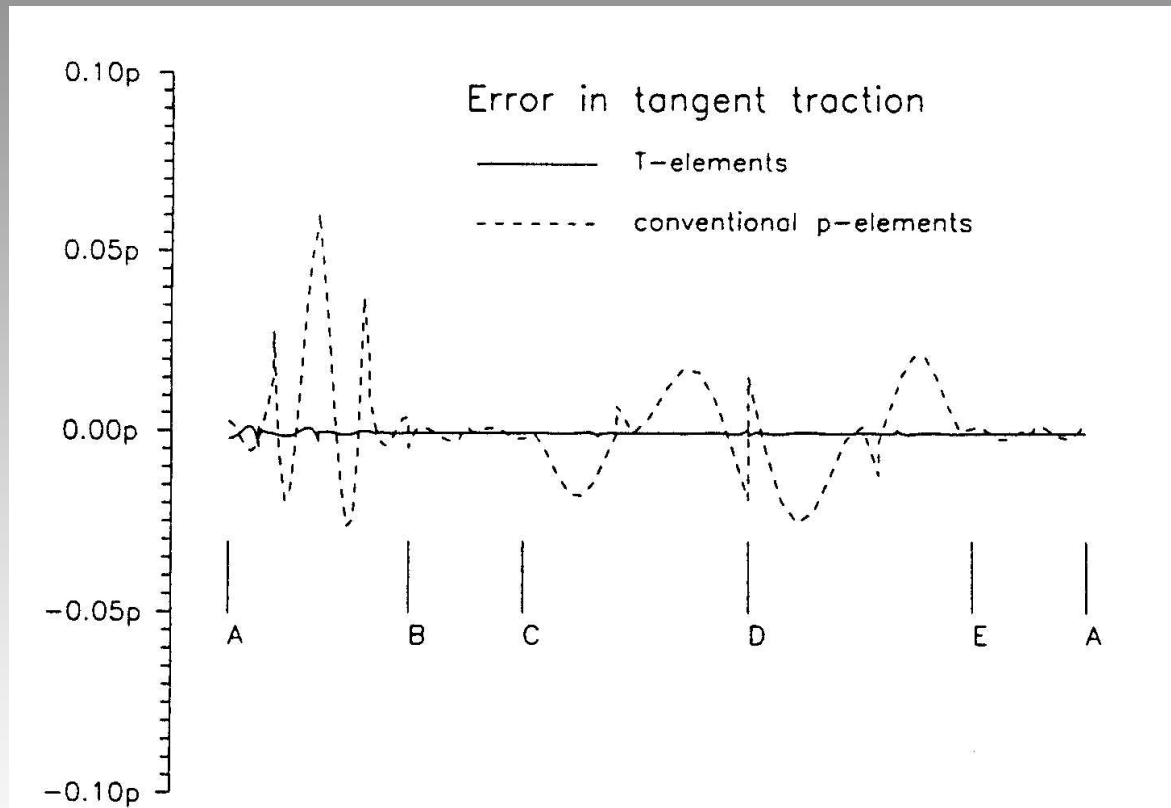
Numerical example: compression of a perforated panel



Numerical example: compression of a perforated panel



Numerical example: compression of a perforated panel

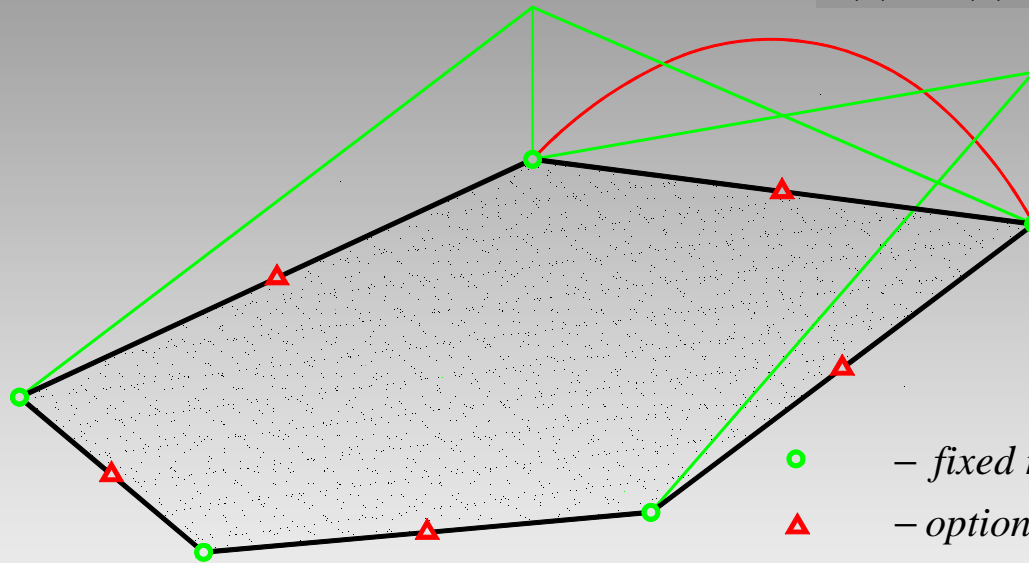


Hybrid-Trefftz displacement p -element (HT-D)

J. Jirousek 1977-1998

$$\begin{aligned} \tilde{N}_1^m &= 1 - \xi^2 \\ \tilde{N}_2^m &= \xi(1 - \xi^2) \\ \tilde{N}_3^m &= \xi^2(1 - \xi^2) \\ &\dots \end{aligned}$$

$$\tilde{\mathbf{u}}(\mathbf{x}) = \tilde{\mathbf{N}}(\mathbf{x}) \mathbf{d} \quad \mathbf{x} \in \Gamma^e$$



- – fixed number of DOF
- ▲ – optional number (M) of DOF

$$\mathbf{u}(\mathbf{x}) = \mathbf{u}^p(\mathbf{x}) + \mathbf{N}(\mathbf{x})\mathbf{c} \quad \mathbf{x} \in \Omega^e$$

\mathbf{N} – matrix of Trefftz functions

$$\mathbf{t}(\mathbf{x}) = \mathbf{t}^p(\mathbf{x}) + \mathbf{T}(\mathbf{x})\mathbf{c} \quad \mathbf{x} \in \Gamma^e$$

T-complete system for 2D elasticity

$$\mathbf{u} = \begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{Bmatrix} u^p \\ v^p \end{Bmatrix} + \sum_{j=1}^l \begin{Bmatrix} N_{uj}^m \\ N_{vj}^m \end{Bmatrix} \cdot c_j^m$$

$$\mathbf{N}_{j+1}^m = \begin{Bmatrix} N_{u\ j+1}^m \\ N_{v\ j+1}^m \end{Bmatrix} = \begin{Bmatrix} \operatorname{Re} Z_{1k} \\ \operatorname{Im} Z_{1k} \end{Bmatrix} \quad \mathbf{N}_{j+4}^m = \begin{Bmatrix} \operatorname{Re} Z_{4k} \\ \operatorname{Im} Z_{4k} \end{Bmatrix}$$

$$\mathbf{N}_{j+2}^m = \begin{Bmatrix} \operatorname{Re} Z_{2k} \\ \operatorname{Im} Z_{2k} \end{Bmatrix} \quad Z_{1k} = (3 - \nu)iz^k + (1 + \nu)kiz\bar{z}^{k-1},$$

$$Z_{2k} = (3 - \nu)z^k - (1 + \nu)kz\bar{z}^{k-1},$$

$$\mathbf{N}_{j+3}^m = \begin{Bmatrix} \operatorname{Re} Z_{3k} \\ \operatorname{Im} Z_{3k} \end{Bmatrix} \quad Z_{3k} = (1 + \nu)i\bar{z}^k,$$

$$Z_{4k} = -(1 + \nu)\bar{z}^k.$$

where: $z = x + iy, \quad \bar{z} = x - iy \quad k = 0, 1, 2, \dots$

(notice: for $k=0$ there are only 2 independent functions)

T-complete system for plate bending problem

$$D \cdot \nabla^4 w = p \qquad w = w^p + \sum_{j=1}^l N_{wj}^b \cdot c_j^b$$

$$N_{j+1}^b = r^2 \cdot \operatorname{Re} z^k$$

$$N_{j+2}^b = r^2 \cdot \operatorname{Im} z^k$$

$$N_{j+3}^b = \operatorname{Re} z^{k+2}$$

$$N_{j+4}^b = \operatorname{Im} z^{k+2}$$

$$k = 0, 1, 2, \dots$$

where:

$$r^2 = x^2 + y^2$$

$$z = x + iy$$

Element of HT-D type

Fitting of internal solution to frame

$$\int_{\Gamma^e} \delta \mathbf{t}^T (\mathbf{u} - \tilde{\mathbf{u}}) d\Gamma = 0$$

$$\delta \mathbf{t}^T = \delta \mathbf{c}^T \mathbf{T}^T \Rightarrow \int_{\Gamma_e} \mathbf{T}^T (\mathbf{u} - \tilde{\mathbf{u}}) d\Gamma = 0 \Rightarrow \mathbf{c} = \mathbf{c}(\mathbf{d})$$

Equivalency of wirtual work

$$\int_{\Gamma^e} \delta \tilde{\mathbf{u}}^T \mathbf{t} d\Gamma = \int_{\Gamma_t^e} \delta \tilde{\mathbf{u}}^T \bar{\mathbf{t}} d\Gamma + \delta \mathbf{d}^T \mathbf{r}$$

$$\mathbf{r} - \mathbf{r}^p = \mathbf{k} \mathbf{d}$$

$$\mathbf{k} = \mathbf{G}^T \mathbf{H}^{-1} \mathbf{G}$$

\mathbf{k} – symmetric stiffnes matrix

\mathbf{d} – vector of degrees of freedom

Element of HT-LS type

Fitting of internal solution to frame

$$\int_{\Gamma^e} (\mathbf{u} - \tilde{\mathbf{u}})^2 d\Gamma \xrightarrow{\mathbf{u}} \min \quad \mathbf{c} = \mathbf{c}(\mathbf{d})$$

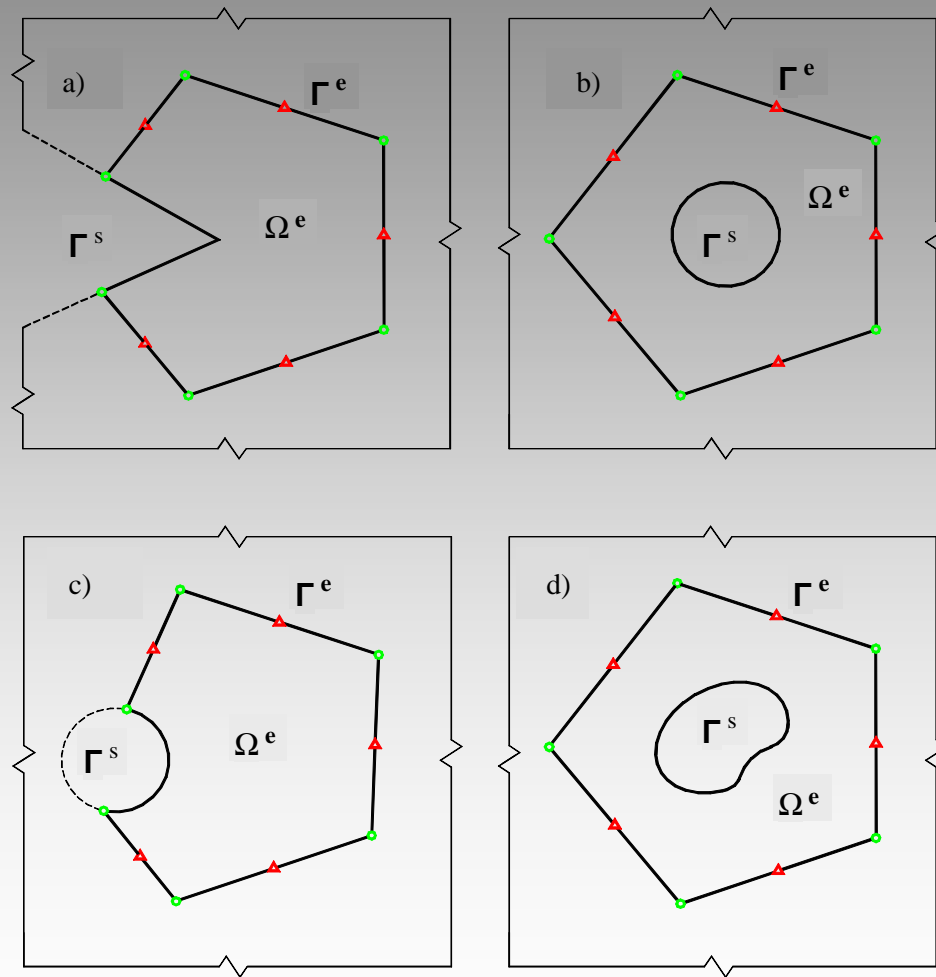
Equivalency of virtual work

$$\int_{\Gamma^e} \delta \mathbf{u}^T \mathbf{t} d\Gamma = \int_{\Gamma^t} \delta \mathbf{u}^T \bar{\mathbf{t}} d\Gamma + \delta \mathbf{d}^T \mathbf{r} \quad \mathbf{r} - \mathbf{r}^p = \mathbf{k} \mathbf{d}$$
$$\mathbf{k} = \mathbf{F}^T \mathbf{H} \mathbf{F}$$

\mathbf{k} – symmetric stiffness matrix

\mathbf{d} – vector of degrees of freedom

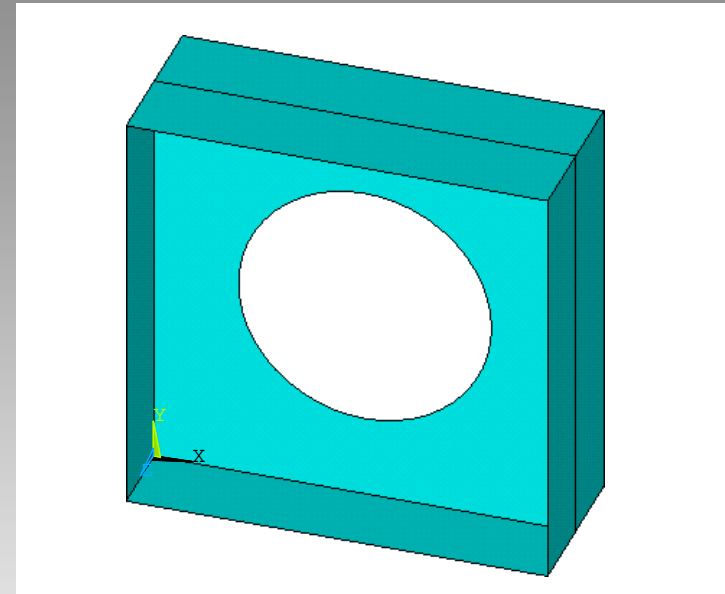
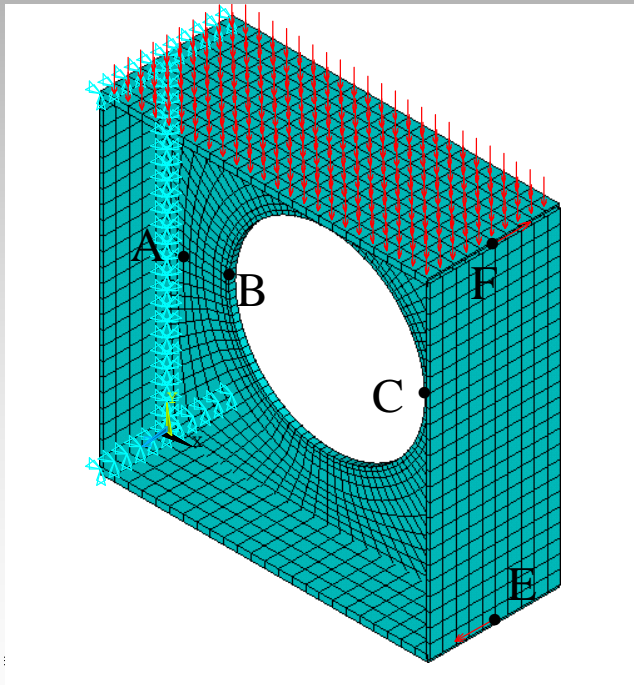
Different types of special purpose T-elements



Folded plate structure — test for T-element features

T-elements: $N_{ACT}=456$

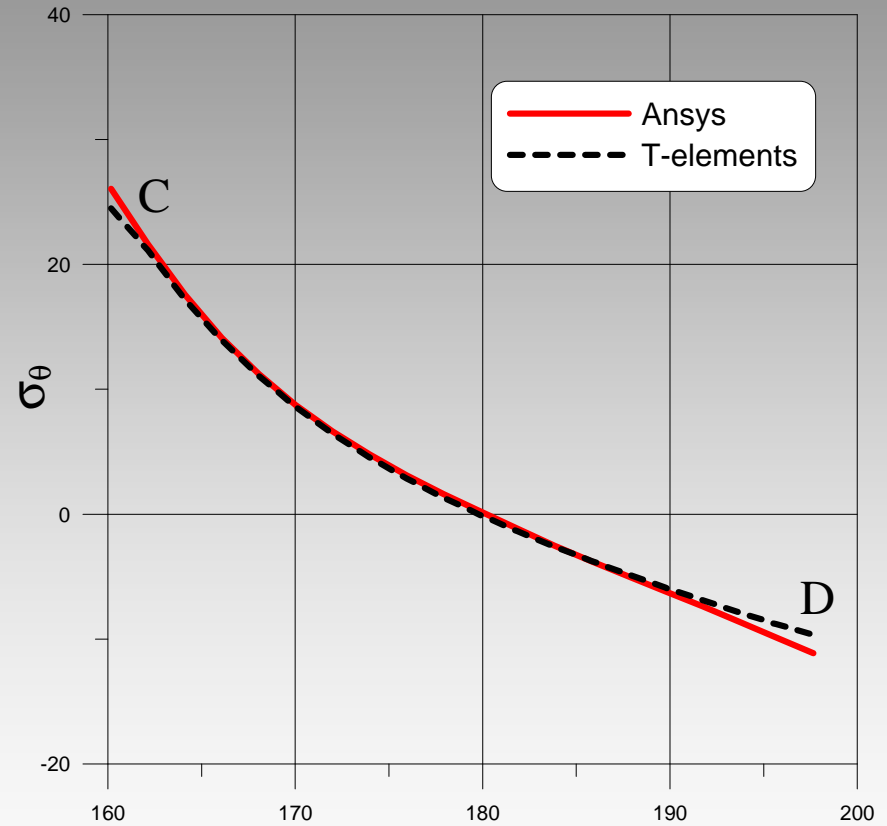
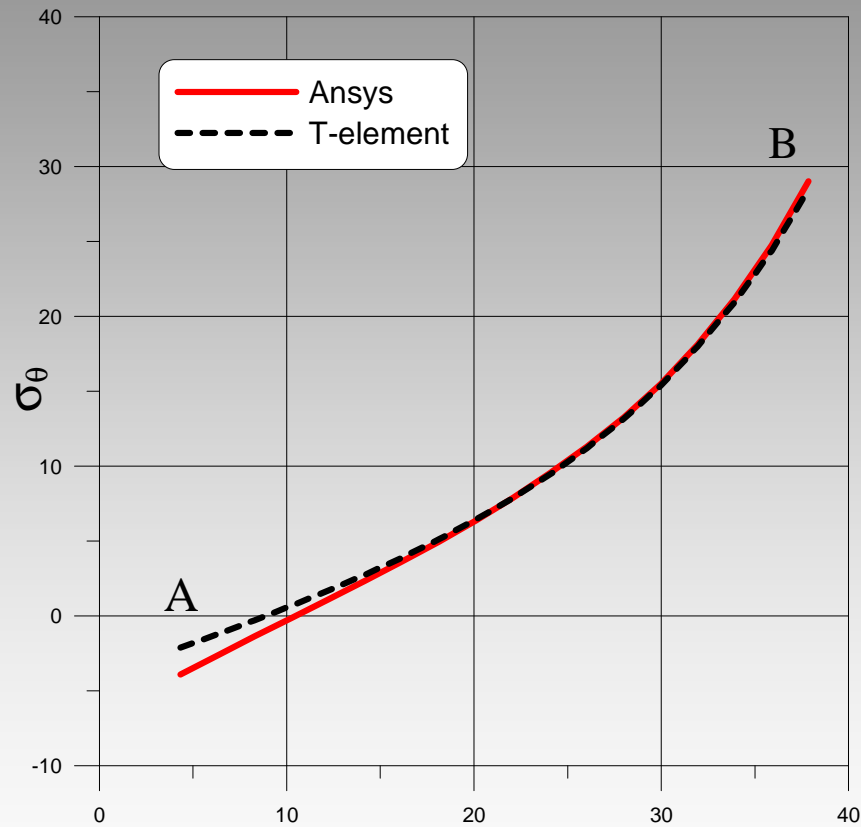
ANSYS: $N_{ACT}=11981$



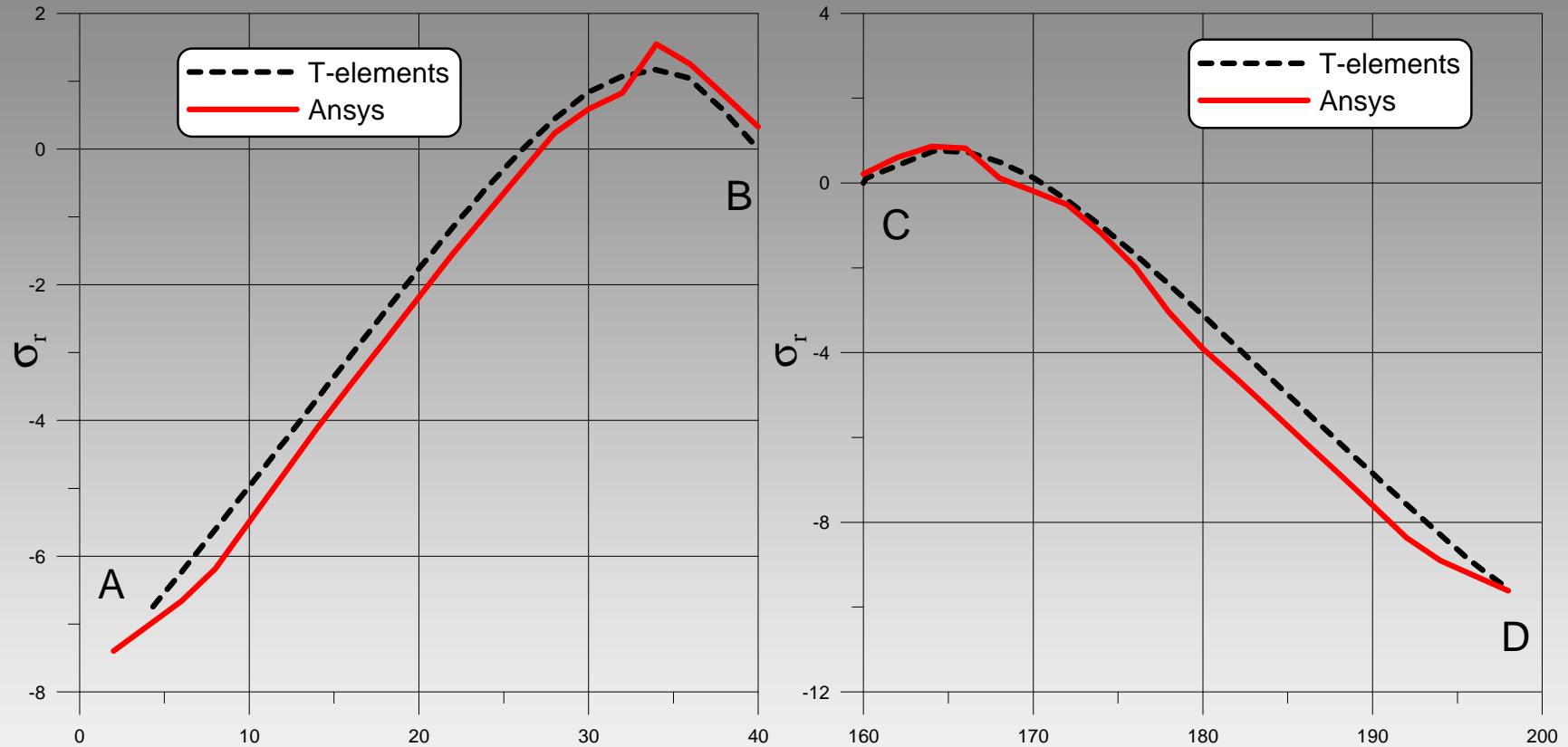
Loadings:
external pressure and nodal forces:

- on top panel, $p_y = -0.5 [N/mm^2]$
- on bottom panel, $p_y = 0.5 [N/mm^2]$
- in point E, $F_x = -100 [N]$, $F_z = 2500 [N]$
- in point F, $F_x = 100 [N]$, $F_z = -2500 [N]$

Results of tests for HT-D element - distribution of circumferential stresses σ_θ along lines AB and CD



Results of tests for HT-D element - radial stresses σ_r along lines AB, CD



Max. and min. value of radial stress σ_r in [MPa] along the boundary of the hole

	ANSYS	T-elements
σ_{rr}^{\min}	-0.5428	$\times 10^{-13}$
σ_{rr}^{\max}	0.4209	$\times 10^{-13}$

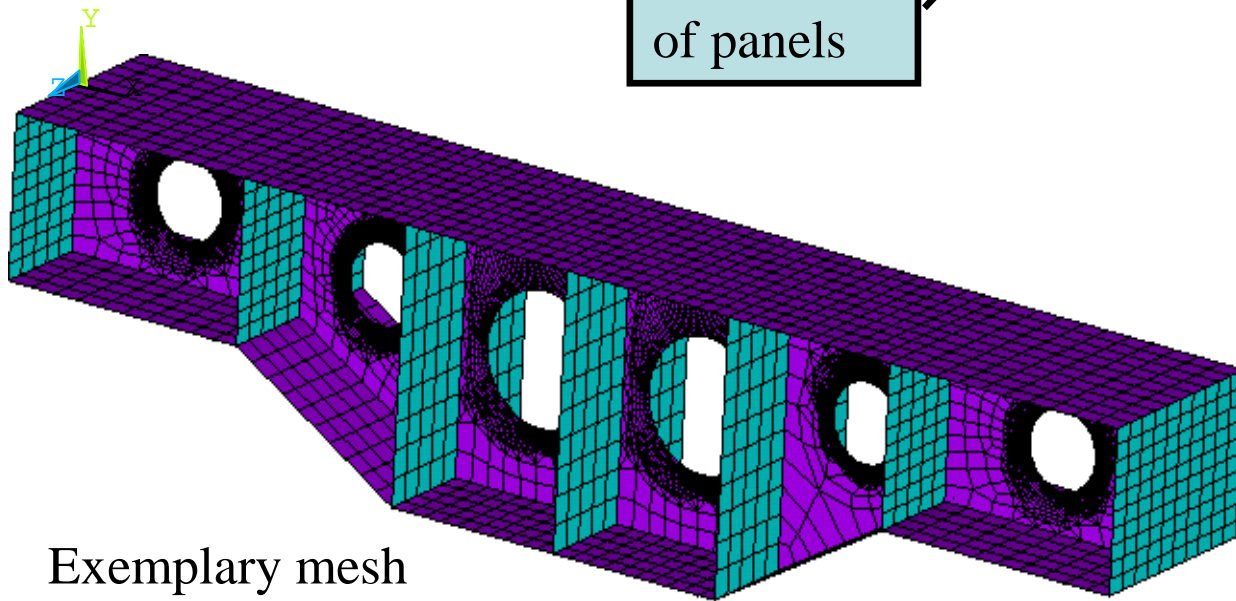
Simply supported at both ends plate girder with 6 circular openings

1st section of panels

2nd section of panels

3rd section of panels

Mesh of T-elements



Exemplary mesh
in ANSYS

Local mesh
refinement introduced
in the vicinity of holes



Thank you for your attention !